

2019-06-25

Weakly nonlinear theory for a gate-type curved array in waves

Michele, Simone

<http://hdl.handle.net/10026.1/17505>

10.1017/jfm.2019.223

Journal of Fluid Mechanics

Cambridge University Press (CUP)

All content in PEARL is protected by copyright law. Author manuscripts are made available in accordance with publisher policies. Please cite only the published version using the details provided on the item record or document. In the absence of an open licence (e.g. Creative Commons), permissions for further reuse of content should be sought from the publisher or author.

Weakly nonlinear theory for a gate-type curved array in waves

S. Michele^{1,†}, E. Renzi¹ and P. Sammarco²

¹Department of Mathematical Sciences, Loughborough University, Leicestershire LE11 3TU, UK

²Department of Civil Engineering and Computer Science, Università degli Studi di Roma ‘Tor Vergata’,
Via del Politecnico 1, 00133 Roma, Italy

(Received 6 September 2018; revised 19 January 2019; accepted 16 March 2019)

We analyse the effect of gate surface curvature on the nonlinear behaviour of an array of gates in a semi-infinite channel. Using a perturbation-harmonic expansion, we show the occurrence of new detuning and damping terms in the Ginzburg–Landau evolution equation, which are not present in the case of flat gates. Unlike the case of linearised theories, synchronous excitation of trapped modes is now possible because of interactions between the wave field and the curved boundaries at higher orders. Finally, we apply the theory to the case of surging wave energy converters (WECs) with curved geometry and show that the effects of nonlinear synchronous resonance are substantial for design purposes. Conversely, in the case of subharmonic resonance we show that the effects of surface curvature are not always beneficial as previously thought.

Key words: coastal engineering, wave–structure interactions

1. Introduction

We examine the nonlinear resonant excitation of an array of curved oscillating gates in a semi-infinite channel. The gate model is similar to that shown in Mei, Stiassnie & Yue (2005), though here we consider a generalised weak horizontal displacement of the gate wetted surface about the vertical plane and the presence of a linear damper exerting a force proportional to the gate velocity. This gives rise to new terms in the evolution equation and to richer dynamics than the case of flat undamped gates.

The array spans the entire channel width, thus the model presented here allows simulation of an infinite periodic array of gates as well (Linton & McIver 2001; Li & Mei 2003). This particular configuration admits eigenfrequencies and corresponding homogeneous trapped-mode solutions at the leading order (Mei *et al.* 1994). Physically, trapped modes of a gate barrier are described by unforced, self-sustained fluid oscillations that decay exponentially from the gates and that do not lose energy through wave radiation (Mei *et al.* 2005). We remark that the wave trapping phenomenon is of considerable interest in several applications such as acoustic resonance in pipes (Hein & Koch 2008), gratings (Porter & Evans 1999), thin elastic

[†] Email address for correspondence: s.michele@lboro.ac.uk

plates (Porter 2007), waveguides (Callan, Linton & Evans 1991; Linton & Ratcliffe 2004), open channels (Evans & Linton 1991), cylinders in a channel (Evans & Porter 1997; Utsunomya & Taylor 1997), edge waves (Blondeaux & Vittori 1995; Li 2007), Venice gates (Li & Mei 2003; Sammarco, Michele & d'Errico 2013), internal waves (Kuznetsov 1993; Nazarov & Videman 2009) and wave energy conversion Michele, Sammarco & d'Errico (2018b).

Linear resonance of trapped modes of a gate barrier in a channel by normally incident waves is not possible because of orthogonality between the modal matrix and the forcing terms (Adamo & Mei 2005); hence we need to extend the analysis to weakly nonlinear waves by taking into account higher-order effects. Similar considerations apply to edge waves (Guza & Bowen 1976; Rockliff 1978), resonance of trapped surface waves around a fixed cylinder (Li & Mei 2006), cross-waves (Lichter & Chen 1987), Faraday waves (Miles 1984; Holmes 1986; Gu & Sethna 1987; Miles & Henderson 1990) and trapped modes for the Venice gates (Mei *et al.* 1994; Vittori, Blondeaux & Seminara 1996; Sammarco, Tran & Mei 1997b; Sammarco *et al.* 1997a).

Here we extend the previous theories by coupling nonlinearity with a small horizontal deviation of the gate surface with respect to the channel depth. We show that this small horizontal deviation of the gate surface produces significant changes in the behaviour of the system. The method of solution adopted here is similar to that used for the Bragg scattering phenomenon of incident waves by bottom ripples (Kirby 1986; Mei, Hara & Naciri 1988; Mei *et al.* 2005; Alam, Liu & Yue 2010). Using perturbation-harmonic expansion up to the third order, we decompose the nonlinear governing equations in a sequence of linear boundary-value problems of order n and harmonic m (Mei *et al.* 2005; Jordan & Smith 2011). Gate shape effects give secular terms for the first harmonic at the second order, so that three timing with a slow time scale and a super-slow time scale is necessary (Nayfeh & Mook 1995).

First, we investigate the synchronous resonance mechanism by monochromatic waves with small amplitude and frequency corresponding to the eigenfrequency of the trapped mode. Note that synchronous excitation is not possible for flat gates, because in that case the evolution equation would be damped and unforced. Here, on the contrary, we obtain products between the gate shape function and the second-order terms that force the first harmonic at the third order. We derive the corresponding complex nonlinear evolution equation of the Ginzburg–Landau form (Aranson & Kramer 2002; Drazin 2002), which describes the time evolution of the resonated trapped mode. Such an equation is more complicated than that already studied by Sammarco *et al.* (1997b,a) and Michele *et al.* (2018b), because it includes new additional terms depending on the shape of the array.

Finally, we apply the theory to the case of an array of surge-type wave energy converters (WECs) with curved geometry. Such bodies oscillate under the action of incident waves and are capable of absorbing energy with potentially large efficiency. For an extensive review we refer to Babarit *et al.* (2012), Dias *et al.* (2017) and Babarit (2018). The literature on the hydrodynamic behaviour of gate-type devices is vast and deeply developed by several authors (Linton & McIver 2001; Mei *et al.* 2005; Renzi & Dias 2012, 2013, 2014; Sammarco *et al.* 2013; Michele *et al.* 2015; Noad & Porter 2015; Sarkar, Renzi & Dias 2015; Michele, Sammarco & d'Errico 2016a,b). Experimental campaigns (Folley, Whittaker & van't Hoff 2007; Henry *et al.* 2010; Wilkinson *et al.* 2017) and numerical investigations (Renzi *et al.* 2014; Schmitt & Elsaesser 2015) are available as well. However, a large part of the theoretical models developed so far on the dynamics of surging WECs neglects nonlinear contributions.

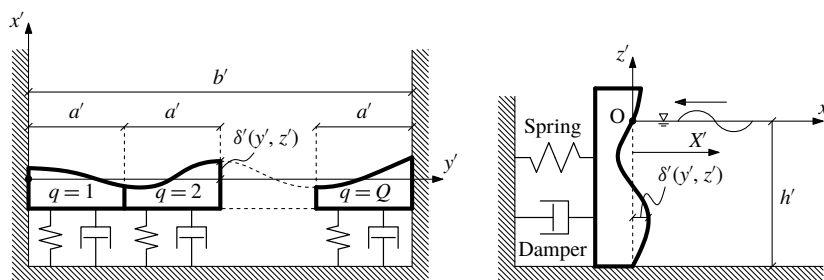


FIGURE 1. Plan geometry of the system and side view of the gate in physical variables.

This can be unjustified when trapped modes are resonantly excited by incident waves. Indeed, Michele *et al.* (2018*b*) showed that subharmonic resonance and mode competition of trapped natural modes significantly increase energy production of a system of gate-type devices. Furthermore, recent investigations on curved flap-type gates suggest that using curved structure could further improve the economics of WECs by maximising wave power extraction in non-resonant configurations (Hodge *et al.* 2017; Michele, Renzi & Sammarco 2018*a*).

We show that nonlinear synchronous resonance of curved WECs yields constructive interactions in terms of generated power that can be significant for design purposes. We remark that this mechanism is not possible for linearised theories or flat WECs. Then we investigate the case of subharmonic resonance and define an optimum criterion to find the power take-off (PTO) coefficient which maximises power extraction. Large efficiency is attained, with the capture factor reaching much greater values than the theoretical maximum of a two-dimensional absorber described by the linear theory (Mei *et al.* 2005). However, in this case the performance of curved gates is sub-optimal with respect to that of flat gates.

2. Governing equations

With reference to figure 1, consider a semi-infinite channel of constant depth h' and width b' . Define a Cartesian reference system (x', y', z') with the x' and y' -axes lying on the undisturbed free surface level and the z' axis pointing upward. Primes indicate physical variables. At $x' = 0$ rests an array of gates, each with mass M' and width a' , allowed to move horizontally (surge) along the channel, under the action of incident harmonic waves. Each gate is connected to the channel back wall by a spring–damper system operating in parallel. The spring has elastic constant C' , while the linear damper has constant damping ν' and exerts a force proportional to the gate velocity. Let us assume incoming waves from $x' \rightarrow +\infty$, normally incident to the gates. Let G_q , $q = 1, \dots, Q$, denote the q th gate and X'_q be the displacement of G_q positive rightward. Then we can define $X'(y', t') = \{X'_1(t'), \dots, X'_q(t'), \dots, X'_Q(t')\}$ as the displacement function of the entire array. The fluid is assumed to be inviscid and incompressible and the flow irrotational. Hence, the velocity potential $\Phi'(x', y', z', t')$ satisfies the Laplace equation in the fluid domain $\Omega'(x', y', z')$. The position of the wetted gate surface is described by

$$x' - X'(y', t') - \delta'(y', z') = 0, \quad (2.1)$$

where δ' denotes the deviation of the array surface from $x' = 0$. Let $A'_T \ll \lambda'$ be the scale of the free-surface trapped oscillations, λ' the wavelength, ω' the eigenfrequency

of the natural mode and g' the acceleration due to gravity. Then introduce the following non-dimensional quantities:

$$\left. \begin{aligned} (x, y, z) &= (x', y', z')/\lambda', & \Phi &= \Phi'/(A'_T \omega' \lambda'), & \zeta &= \zeta'/A'_T, & t &= t' \omega', \\ (a, b, h) &= (a', b', h')/\lambda', & X &= X'/A'_T, & \delta &= \delta'/\delta'_g, & G &= g'/(\omega'^2 \lambda'), \end{aligned} \right\} \quad (2.2)$$

where ζ' is the free-surface elevation, $\delta'_g = O(A'_T)$ the length scale for δ' and G the non-dimensional eigenfrequency. Let the following length ratios be much smaller than unity

$$\epsilon = A'_T/\lambda' \ll 1, \quad \mu = \delta'_g/\lambda' \ll 1, \quad \mu = O(\epsilon). \quad (2.3a-c)$$

Since the derivatives of δ' with respect to the coordinates (y', z') are of order ϵ as well, the latter assumptions imply that the shape of the array must be smooth and regular.

Using the dimensionless variables (2.2)–(2.3), we derive the following governing equations, boundary conditions and equation of motion in non-dimensional form. The Laplace and Bernoulli equations in the fluid domain are, respectively,

$$\nabla^2 \Phi = 0, \quad (2.4)$$

$$-\frac{p'}{\rho' \omega'^2 \lambda'^2} = Gz + \epsilon \Phi_t + \epsilon^2 \frac{1}{2} |\nabla \Phi|^2, \quad (2.5)$$

where ρ' is the fluid density, while subscripts denote differentiation with respect to the relevant variable. The dynamic and mixed boundary conditions on the free surface read, respectively,

$$-G\zeta = \Phi_t + \epsilon \frac{1}{2} |\nabla \Phi|^2, \quad z = \epsilon \zeta, \quad (2.6a, b)$$

$$\Phi_{tt} + G\Phi_z + \epsilon |\nabla \Phi|^2_t + \epsilon^2 \frac{1}{2} \nabla \Phi \cdot \nabla |\nabla \Phi|^2 = 0, \quad z = \epsilon \zeta, \quad (2.7)$$

while the no-flux conditions at the bottom and channel walls require

$$\Phi_z = 0, \quad z = -h, \quad (2.8)$$

$$\Phi_y = 0, \quad y = 0 \quad \text{and} \quad y = b. \quad (2.9)$$

The kinematic condition on the array surface

$$x = \epsilon X + \mu \delta, \quad (2.10)$$

can be now written as

$$\Phi_x = X_t + \mu(\Phi_y \delta_y + \Phi_z \delta_z). \quad (2.11)$$

The equation of motion of the q th curved gate coupled with a linear damper is given by

$$\epsilon M X_{q,tt} + \epsilon C G X_q + \epsilon^3 \nu X_{q,t} = \int_{(q-1)a}^{qa} dy \left\{ \int_{-1}^{\epsilon \zeta} dz \left(\epsilon \Phi_t + \epsilon^2 \frac{1}{2} |\nabla \Phi|^2 \right) + \int_0^{\epsilon \zeta} Gz dz \right\}, \quad (2.12)$$

where $M = M'/(\rho' \lambda'^3)$ is the non-dimensional mass of the gate, $C = C'/(g' \rho' \lambda'^2)$ the non-dimensional stiffness of the spring and $\nu = \nu'/(A'^2_T \omega' \rho' \lambda')$ the non-dimensional damping coefficient. In (2.12) we have assumed the contribution due to the damping force on the gate motion to be small if compared to the other terms. Large values of ν' comparable with leading-order terms would render the equation of motion

at $O(1)$ damped and unforced, so that a trapped-mode solution would not be possible anymore (Michele *et al.* 2018b). Assuming $A'_T \sim O(1)$ m, $\omega' \sim O(1)$ rad s⁻¹, $\lambda' \sim O(10)$ – $O(10^2)$ m, the values of ν' that satisfy the scale above should be of order $O(10^4)$ – $O(10^5)$ kg s⁻¹. In § 5 we demonstrate that these orders of magnitude are physically congruent with power take-off systems for practical engineering applications (Mei *et al.* 2005).

The free-surface boundary conditions are evaluated in correspondence of $z = \epsilon\zeta$, thus Taylor expanding (2.6) and (2.7) about $z = 0$ up to $O(\epsilon^2)$ yields, respectively,

$$-G\zeta = [\Phi_t]_{z=0} + \epsilon\zeta[\Phi_{tz}]_{z=0} + \epsilon^2\frac{\zeta^2}{2}[\Phi_{tzz}]_{z=0} + \epsilon\frac{1}{2}[\nabla\Phi]^2_{z=0} + \epsilon^2\frac{\zeta}{2}[\nabla\Phi]_z^2_{z=0}, \quad (2.13)$$

$$\begin{aligned} [\Phi_{tt} + G\Phi_z]_{z=0} + \epsilon\zeta[\Phi_{ttz} + G\Phi_{zz}]_{z=0} + \epsilon^2\frac{\zeta^2}{2}[\Phi_{ttzz} + G\Phi_{zzz}]_{z=0} + \epsilon[\nabla\Phi]_t^2_{z=0} \\ + \epsilon^2\zeta[\nabla\Phi]_{tz}^2_{z=0} + \epsilon^2\frac{1}{2}[\nabla\Phi \cdot \nabla|\nabla\Phi|^2]_{z=0} = 0. \end{aligned} \quad (2.14)$$

Similarly, the kinematic boundary condition on the gate surface can be Taylor expanded about $x = 0$:

$$\begin{aligned} [\Phi_x]_{x=0} = X_t - (\epsilon X + \mu\delta)[\Phi_{xx}]_{x=0} - \frac{1}{2}(\epsilon X + \mu\delta)^2[\Phi_{xxx}]_{x=0} \\ + \mu[\Phi_y\delta_y + \Phi_z\delta_z]_{x=0} + \mu(\epsilon X + \mu\delta)[\Phi_{xy}\delta_y + \Phi_{xz}\delta_z]_{x=0}. \end{aligned} \quad (2.15)$$

Finally the equation of motion (2.12) becomes:

$$\begin{aligned} MX_{q,tt} + GCX_q + \epsilon^2\nu X_{q,t} = \int_{(q-1)a}^{qa} dy \int_{-1}^0 dz \left\{ \Phi_t + \epsilon \left[\Phi_{tx} \left(X + \frac{\mu\delta}{\epsilon} \right) + \frac{|\nabla\Phi|^2}{2} \right] \right. \\ \left. + \epsilon^2 \left(X + \frac{\mu\delta}{\epsilon} \right) \left[\frac{\Phi_{txx}}{2} \left(X + \frac{\mu\delta}{\epsilon} \right) + \frac{|\nabla\Phi|_x^2}{2} \right] \right\}_{x=0} + \epsilon \int_{(q-1)a}^{qa} dy \left\{ \left[\frac{G\zeta^2}{2} + \Phi_t\zeta \right] \right. \\ \left. + \epsilon \left[(G\zeta\zeta_x + \Phi_{tx}\zeta) \left(X + \frac{\mu\delta}{\epsilon} \right) + \frac{\Phi_{tz}\zeta^2}{2} + \frac{\zeta|\nabla\Phi|^2}{2} \right] \right\}_{x=0, z=0}. \end{aligned} \quad (2.16)$$

3. Multiple-scale analysis and three timing

Let us introduce the following expansions of the non-dimensional velocity potential, free-surface elevation and gate oscillation:

$$\Phi = \Phi_1(x, y, z, t, t_1, t_2) + \epsilon\Phi_2(x, y, z, t, t_1, t_2) + \epsilon^2\Phi_3(x, y, z, t, t_1, t_2) + O(\epsilon^3), \quad (3.1)$$

$$\zeta = \zeta_1(x, y, t, t_1, t_2) + \epsilon\zeta_2(x, y, t, t_1, t_2) + \epsilon^2\zeta_3(x, y, t, t_1, t_2) + O(\epsilon^3), \quad (3.2)$$

$$X_q = X_{q,1}(t, t_1, t_2) + \epsilon X_{q,2}(t, t_1, t_2) + \epsilon^2 X_{q,3}(t, t_1, t_2) + O(\epsilon^3), \quad (3.3)$$

$$X = X_1(y, t, t_1, t_2) + \epsilon X_2(y, t, t_1, t_2) + \epsilon^2 X_3(y, t, t_1, t_2) + O(\epsilon^3), \quad (3.4)$$

where $t_1 = \epsilon t$ and $t_2 = \epsilon^2 t$ denote two slow time scales of the modal amplitude growth. Unlike the case of Michele *et al.* (2018b), the three-timing assumption is necessary here because of the presence in the governing equations of terms representing the shape of the array. These terms correspond to a resonant forcing for the first-harmonic solution at the second order, so that a solvability condition at $O(\epsilon)$ must be applied in order to avoid secularity. The three time scales add terms in the evolution equation for

the modal amplitude at the third order $O(\epsilon^2)$ and the corresponding stability analysis for the equilibrium states increases in complexity. Usage of the expansions (3.1)–(3.4) for the unknowns yields for $n = 1, 2, 3$, the following equations.

Laplace's equation:

$$\nabla^2 \Phi_n = 0, \quad \text{in } \Omega. \quad (3.5)$$

Free-surface dynamic condition:

$$-G\zeta_n = \Phi_{n_t} + \mathcal{B}_n, \quad z = 0, \quad (3.6a, b)$$

where

$$\mathcal{B}_1 = 0, \quad \mathcal{B}_2 = \Phi_{1_{t_1}} + \Phi_{1_{tz}} \zeta_1 + \frac{1}{2} |\nabla \Phi_1|^2, \quad (3.7a, b)$$

$$\mathcal{B}_3 = \Phi_{1_{t_2}} + \Phi_{2_{t_1}} + \Phi_{1_{t_1 z}} \zeta_1 + \Phi_{1_{tz}} \zeta_2 + \Phi_{2_{tz}} \zeta_1 + \frac{\Phi_{1_{tzz}} \zeta_1^2}{2} + \nabla \Phi_1 \cdot \nabla \Phi_2 + \frac{\zeta_1}{2} |\nabla \Phi_1|_{tz}^2. \quad (3.8)$$

Free-surface mixed condition:

$$\Phi_{n_{tt}} + G\Phi_{n_z} = \mathcal{F}_n, \quad z = 0, \quad (3.9)$$

where

$$\mathcal{F}_1 = 0, \quad \mathcal{F}_2 = -2\Phi_{1_{tt_1}} - \zeta_1(\Phi_{1_{tzz}} + G\Phi_{1_{zz}}) - |\nabla \Phi_1|_t^2, \quad (3.10a, b)$$

$$\begin{aligned} \mathcal{F}_3 = & -2\Phi_{1_{tt_2}} - 2\Phi_{2_{tt_1}} - \Phi_{1_{t_1 t_1}} - 2\Phi_{1_{t_1 z}} \zeta_1 - \zeta_2(\Phi_{1_{tzz}} + G\Phi_{1_{zz}}) - \zeta_1(\Phi_{2_{tzz}} + G\Phi_{2_{zz}}) \\ & - \frac{\zeta_1^2}{2}(\Phi_{1_{tzz}} + G\Phi_{1_{zz}}) - 2(\nabla \Phi_1 \cdot \nabla \Phi_2)_t - \zeta_1 |\nabla \Phi_1|_{tz}^2 - \frac{1}{2} \nabla \Phi_1 \cdot \nabla |\nabla \Phi_1|^2 \\ & - |\nabla \Phi_1|_{t_1}^2. \end{aligned} \quad (3.11)$$

No-flux boundary condition at the horizontal bottom:

$$\Phi_{n_z} = 0, \quad z = -h. \quad (3.12a, b)$$

No-flux boundary condition on the channel vertical walls:

$$\Phi_{n_y} = 0, \quad y = 0 \quad \text{and} \quad y = b. \quad (3.13)$$

Kinematic condition on the array surface:

$$\Phi_{n_x} = X_{n_t} + \mathcal{G}_n, \quad x = 0, \quad (3.14)$$

where

$$\mathcal{G}_1 = 0, \quad \mathcal{G}_2 = X_{1_{t_1}} - \Phi_{1_{xx}} \left(X_1 + \frac{\mu\delta}{\epsilon} \right) + \frac{\mu}{\epsilon} (\Phi_{1_y} \delta_y + \Phi_{1_z} \delta_z), \quad (3.15a, b)$$

$$\begin{aligned} \mathcal{G}_3 = & X_{1_{t_2}} + X_{2_{t_1}} - \Phi_{1_{xx}} X_2 - \Phi_{2_{xx}} \left(X_1 + \frac{\mu\delta}{\epsilon} \right) - \frac{\Phi_{1_{xxx}}}{2} \left(X_1 + \frac{\mu\delta}{\epsilon} \right)^2 \\ & + \frac{\mu}{\epsilon} (\Phi_{2_y} \delta_y + \Phi_{2_z} \delta_z) + \frac{\mu}{\epsilon} (\Phi_{1_{xy}} \delta_y + \Phi_{1_{xz}} \delta_z) \left(X_1 + \frac{\mu\delta}{\epsilon} \right). \end{aligned} \quad (3.16)$$

Equation of motion of the q th gate:

$$MX_{q,n_t} + GCX_{q,n} = \int_{(q-1)a}^{qa} dy \int_{-1}^0 \Phi_{n_t} dz + \mathcal{D}_n, \quad (3.17)$$

where

$$\mathcal{D}_1 = 0, \quad (3.18)$$

$$\begin{aligned} \mathcal{D}_2 = & -2MX_{1_{n_1}} + \int_{(q-1)a}^{qa} dy \left\{ G \frac{\zeta_1^2}{2} + \Phi_{1_t} \zeta_1 \right\}_{x=0, z=0} \\ & + \int_{(q-1)a}^{qa} dy \int_{-1}^0 dz \left\{ \Phi_{1_{t_1}} + \Phi_{1_{tx}} \left(X_1 + \frac{\mu\delta}{\epsilon} \right) + \frac{1}{2} |\nabla \Phi_1|^2 \right\}_{x=0}, \end{aligned} \quad (3.19)$$

$$\begin{aligned} \mathcal{D}_3 = & -M(2X_{1_{n_2}} + 2X_{2_{n_1}} + X_{1_{t_1 t_1}}) + \int_{(q-1)a}^{qa} dy \left\{ G \zeta_1 \zeta_2 + \frac{\Phi_{1_{tx}} \zeta_1^2}{2} + \Phi_{1_t} \zeta_2 + \Phi_{2_t} \zeta_1 \right. \\ & + \Phi_{1_{t_1}} \zeta_1 + \Phi_{1_{tx}} \zeta_1 \left(X_1 + \frac{\mu\delta}{\epsilon} \right) + \frac{\zeta_1}{2} |\nabla \Phi_1|^2 + G \zeta_1 \zeta_{1_x} \left(X_1 + \frac{\mu\delta}{\epsilon} \right) \left. \right\}_{x=0, z=0} \\ & + \int_{(q-1)a}^{qa} dy \int_{-1}^0 dz \left\{ \Phi_{1_{t_2}} + \Phi_{2_{t_1}} + \Phi_{1_{tx}} X_2 + \nabla \Phi_1 \cdot \nabla \Phi_2 + \frac{\Phi_{1_{txx}}}{2} \left(X_1 + \frac{\mu\delta}{\epsilon} \right)^2 \right. \\ & + \left. \left(\Phi_{2_{tx}} + \Phi_{1_{t_1 x}} + \frac{1}{2} |\nabla \Phi_1|^2_x \right) \left(X_1 + \frac{\mu\delta}{\epsilon} \right) \right\}_{x=0} - \nu X_{q, 1_t}. \end{aligned} \quad (3.20)$$

Because of harmonic motion, higher-order solutions imply higher harmonics (Jordan & Smith 2011). Hence we return to physical variables except for t_1 and t_2 , omit the primes for convenience and assume the following harmonic expansions of the unknowns:

$$\{\Phi_n, \zeta_n, X_{q,n}, X_n\} = \sum_{m=0}^n \{\phi_{nm}, \eta_{nm}, \chi_{q,nm}, \chi_{nm}\} e^{-im\omega t} + *, \quad (3.21)$$

where the symbol $*$ indicates the complex conjugate of the terms inside the series. Substitution of the latter expansion into the governing equation and boundary conditions allows us to split the nonlinear problem in a sequence of linear boundary-value problems of order n and harmonic m :

$$\nabla^2 \phi_{nm} = 0, \quad \text{in } \Omega, \quad (3.22)$$

$$\phi_{nm_z} = \phi_{nm} \frac{m^2 \omega^2}{g} + \mathcal{F}_{nm}, \quad z = 0, \quad (3.23)$$

$$\eta_{nm} = \phi_{nm} \frac{im\omega}{g} + \mathcal{B}_{nm}, \quad z = 0, \quad (3.24)$$

$$\phi_{nm_z} = 0, \quad z = -h, \quad (3.25)$$

$$\phi_{nm_y} = 0, \quad y = 0, y = b, \quad (3.26)$$

$$\phi_{nm_x} = -im\omega \chi_{nm} + \mathcal{G}_{nm}, \quad x = 0. \quad (3.27)$$

The latter must be solved jointly with the equation of motion for each gate G_q (3.17), which now becomes:

$$-m^2\omega^2 M\chi_{q,nm} + C\chi_{q,nm} = -im\omega\rho \int_{(q-1)a}^{qa} dy \int_{-h}^0 \phi_{nm} dz + \mathcal{D}_{nm}, \quad (3.28)$$

where the forcing terms \mathcal{F}_{nm} , \mathcal{B}_{nm} , \mathcal{G}_{nm} and the \mathcal{D}_{nm} values are defined for each order n and harmonic m in appendix A. Having obtained the governing equations at the different orders, we are now in a position to investigate the nonlinear synchronous excitation of trapped modes.

4. Nonlinear synchronous excitation of a single trapped mode

In this section we analyse the synchronous excitation of a single trapped mode by small incident waves at the second order $O(\epsilon)$. We remark that such nonlinear dynamics is peculiar to curved gates and nonlinear resonance mechanisms. Indeed, trapped modes cannot be resonated in linear theories by normally incident waves because of orthogonality between the modal matrix and forcing terms, while flat-gate systems do not allow nonlinear synchronous excitation when the incident waves are small compared to the trapped wave field. This is due to the absence of first-harmonic terms at the third order that include the forcing incident wave potential. In that case, the evolution equation would be damped and unforced and the corresponding solution would be given by the trivial stable state.

As in Michele *et al.* (2018b), we perform an asymptotic analysis up to the third order $O(\epsilon^2)$. For the sake of brevity, we show the solutions of each boundary-value problem of order n and harmonic m in appendix A. We obtain that the second-order inhomogeneous problem is forced by products between the first-order solution ϕ_{11} and the gate shape function δ . Since ω and ϕ_{11} solve the homogeneous first-harmonic problem at the leading order $O(1)$ (see § A.2), a solvability condition must be now applied to ϕ_{11} and ϕ_{21} to avoid secularity. Green's theorem over the entire fluid domain Ω yields

$$\iiint_{\Omega} (\phi_{11} \nabla^2 \phi_{21} - \phi_{21} \nabla^2 \phi_{11}) d\Omega = \iint_{\partial\Omega} \left(\phi_{11} \frac{\partial \phi_{21}}{\partial n} - \phi_{21} \frac{\partial \phi_{11}}{\partial n} \right) dS = 0, \quad (4.1)$$

where the normal n points outward the volume boundaries $\partial\Omega$. By performing straightforward algebra we obtain the evolution equation

$$\chi_{t_1} - \frac{ic_{\delta}}{\omega\epsilon} \chi = 0, \quad (4.2)$$

where the coefficient c_{δ} is real and given by

$$c_{\delta} = \frac{1}{c_f} \int_0^b dy \int_{-h}^0 dz \{ f_{11} (f_{11,xx} \delta - f_{11,y} \delta_y - f_{11,z} \delta_z) + r\omega \delta f_{11,x} \}, \quad (4.3)$$

in which

$$c_f = \int_0^b dy \int_0^{+\infty} \frac{2\omega f_{11}^2}{g} dx + \int_0^b dy \int_{-h}^0 2f_{11} r dz + \sum_{q=1}^Q \frac{2a\omega M r_q^2}{\rho}. \quad (4.4)$$

In the previous expressions, f_{11} represents the spatial dependence of the first-order trapped velocity potential (A 3), while r_q is the modal shape given by the solution of the equation of motion at the leading order (see also Michele *et al.* 2018b). The complex evolution equation (4.2) is linear, so the corresponding solution is readily given by

$$\chi(t_1, t_2) = \vartheta(t_2)e^{-(ic_\delta t_1/\omega\epsilon)} = \vartheta(t_2)e^{-ic_\delta t}. \quad (4.5)$$

Thus, the coefficient c_δ represents a modulation of the modal amplitude growth given by gate curvature. On the other hand, for flat gates ($\delta=0$) the coefficient $c_\delta=0$ and χ depends on the super-slow time scale t_2 only (Sammarco *et al.* 1997b; Michele *et al.* 2018b). We point out that curved gates do not always give $c_\delta \neq 0$. For example, anti-symmetrical shapes with respect to the vertical plane $y=b/2$ yield $c_\delta=0$.

At the third order, the inhomogeneous problem is forced by second-order and first-order solutions, respectively. For the same reasons of the first-harmonic problem at the second order, analysed before, we invoke the solvability condition by applying Green's theorem to ϕ_{11} and ϕ_{31} over Ω . After some lengthy algebra we obtain

$$\int_0^b dy \int_0^{+\infty} f_{11} \mathcal{F}_{31} dx - \int_0^b dy \int_{-h}^0 f_{11} \mathcal{G}_{31} dz + \sum_{q=1}^Q \frac{i\mathcal{D}_{31}r_q}{\rho} = 0. \quad (4.6)$$

Combination of the latter terms with the shape function δ and its derivatives gives the following evolution equation of the Ginzburg–Landau type (Drazin 2002) for the modal amplitude depending on the slow time scale t_2 :

$$-\epsilon^2 i \vartheta_{t_2} = \vartheta(c_A + ic_B) + \vartheta^2 \vartheta^*(c_N + ic_R) + Ae^{(-ic_\delta t_1/\epsilon)}(c_S + ic_U) + i\vartheta \nu c_L, \quad (4.7)$$

where ϑ^* is the complex conjugate of ϑ . The expressions for the real coefficients c_A , c_B , c_N , c_R , c_S , c_U , c_L in (4.7) are quite long and can be found in appendix B. The latter equation now has additional terms when compared to the evolution equation in Michele *et al.* (2018b) for flat WECs, or the evolution equation for Venice gates in Sammarco *et al.* (1997b). These are the new terms c_A , c_B , the complex forcing coefficient $(c_S + ic_U)$ and the real coefficient c_δ given by (4.3).

The coefficients c_A and c_B represent, respectively, detuning and damping caused by the shape of the array. Flat gates ($\delta=0$) give $c_A=c_B=0$. Note also that c_A and c_B are invariant for profiles that are symmetric about $x=0$. In other words $[c_A(\delta), c_B(\delta)] = [c_A(-\delta), c_B(-\delta)]$. Concerning the other coefficients, c_N represents nonlinearity, c_R is the radiation damping due to the second-harmonic radiation at the second order, c_S and c_U represent the energy influx by the incident waves, while c_L represents the effects due to the linear damper.

Instead of perfect resonance, we consider a detuning $\Delta\omega$ between the trapped mode and the incident waves such that the ratio $\Delta\omega/\omega = \omega_2\epsilon^2$ (Sammarco *et al.* 1997b). Then, the evolution equation becomes

$$-i\vartheta_{t_2} = \vartheta(c_A + ic_B) + \vartheta^2 \vartheta^*(c_N + ic_R) + Ae^{-i(\omega_2 t_2 + (c_\delta t_1/\epsilon))}(c_S + ic_U) + i\vartheta \nu c_L. \quad (4.8)$$

After the following change of variables

$$\vartheta = \bar{\vartheta} e^{-i(\omega_2 t_2 + (t_1 c_\delta/\omega\epsilon))}, \quad (4.9)$$

we get

$$-i\bar{\vartheta}_t = \bar{\vartheta}(\Delta\omega + c_A + ic_B) + \bar{\vartheta}^2 \bar{\vartheta}^*(c_N + ic_R) + A(c_S + ic_U) + i\bar{\vartheta} \nu c_L. \quad (4.10)$$

By multiplying by $\bar{\vartheta}^*$ both sides of (4.10) and subtracting its complex conjugate we obtain the energy equation for synchronous excitation:

$$\frac{d|\bar{\vartheta}|^2}{dt} = -2(c_B + \nu c_L)|\bar{\vartheta}|^2 - 2c_R|\bar{\vartheta}|^4 - 2A \operatorname{Im}\{(c_S - ic_U)\bar{\vartheta}\}. \quad (4.11)$$

Since c_R and the term $c_B + \nu c_L$ are both positive, their effect is to damp the modal amplitude.

Now define $\bar{\vartheta}$ in action-angle variables form, i.e. $\bar{\vartheta} = Re^{i\psi}$. Then, from (4.10) we obtain the following system of two real differential equations:

$$\left. \begin{aligned} R_t &= -R(c_L\nu + c_B) - R^3c_R - A(c_U \cos \psi - c_S \sin \psi), \\ \psi_t &= \Delta\omega + R^2c_N + \frac{A}{R}(c_S \cos \psi + c_U \sin \psi). \end{aligned} \right\} \quad (4.12)$$

The trivial fixed point does not exist, while non-trivial fixed points related to unstable and stable equilibria correspond to the roots of the equation

$$-R(c_L\nu + c_B + R^2c_R) + \sqrt{A^2(c_S^2 + c_U^2) - R^2(c_N R^2 + \Delta\omega + c_A)^2} = 0. \quad (4.13)$$

The latter equation admits either a single stable fixed point or the coexistence of three roots, i.e. two stable points and one unstable saddle. This depends on the values of both detuning $\Delta\omega$ and damping coefficient ν and will be investigated in the next section. Differently from what was obtained by Sammarco *et al.* (1997b) and Michele *et al.* (2018b), the stable branch never coincides with the origin, hence the amplitude at the equilibrium is always positive.

5. Application to surge-type wave energy converters

The theory developed so far is now applied to surge-type WECs. Here, the linear damper represents a power take-off (PTO) system with damping coefficient $\nu = \nu_{pto}$ (Mei *et al.* 2005). Let us consider the simplest case of $Q = 2$ gates. Solution of the $O(1)$ problem yields a single out-of-phase trapped mode with eigenvector $r_q = \{1, -1\}$. The water depth is $h = 5$ m and the gate width is $a = 5$ m. The amplitude of the incident waves must be at the second order, thus we assume $A = 0.1$ m. Since we require that $\nu_{pto} \sim O(A_T^2 \omega \rho \lambda)$ (see § 2), if $\omega \sim O(1)$ rad s⁻¹ and the wavelength $\lambda \sim O(10) - O(10^2)$ m, the numerical values of ν_{pto} that satisfy the scale above ought to be of order $O(10^4) - O(10^5)$ kg s⁻¹. To prove that this range is reasonably valid for practical engineering applications, let us maximise power extraction when a flat gate moves in phase. Solution of the linearised two-dimensional radiation velocity potential yields the following expression for the radiation damping

$$\bar{\nu} = \frac{\rho \omega a \sinh^2 k_0 h}{k_0^3} \left(\frac{2k_0 h + \sinh 2k_0 h}{4k_0} \right)^{-1}, \quad (5.1)$$

where k_0 is the solution of the dispersion relation $\omega^2 = gk_0 \tanh k_0 h$. Maximum power extraction efficiency requires resonance and $\nu_{pto} = \bar{\nu}$ (Mei *et al.* 2005), hence the latter expression gives a first estimate for the optimal damping which maximises the generated power for a surging gate. For the array dimensions considered here, expression (5.1) yields $\bar{\nu} \sim O(10^5)$ kg s⁻¹ when $\omega \sim 1$ rad s⁻¹, i.e. a value that

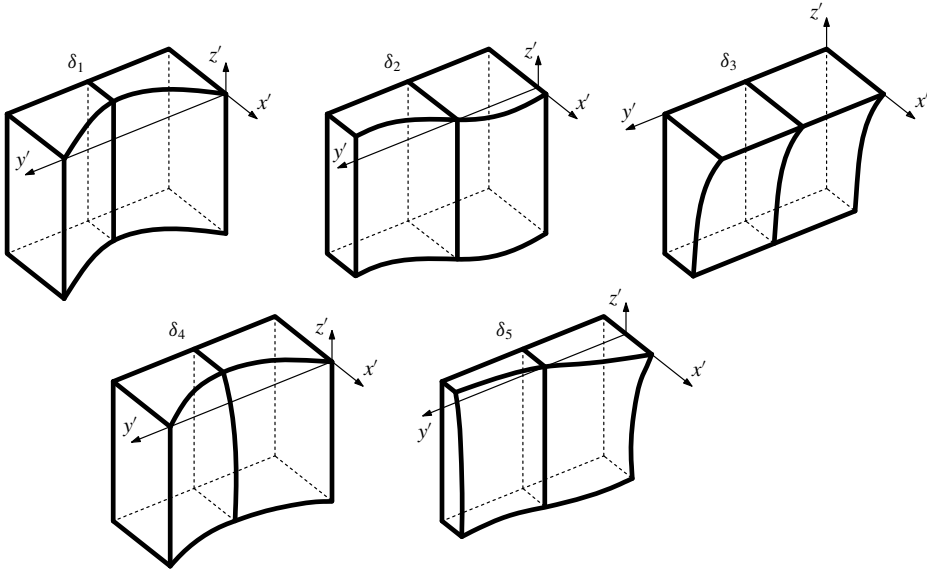


FIGURE 2. Sketch of the five configurations defined by expressions (5.2).

matches the damping coefficient scaling assumed in this work. This means that nonlinear effects due to hydrodynamic contributions can be important, because they are comparable to the PTO damping term. Thus, neglecting nonlinear hydrodynamic terms might cause one to overlook constructive resonance phenomena like the synchronous resonance mechanism analysed here.

Now, let us compare the flat gate ($\delta = 0$) with five different gate configurations that can be of practical engineering interest, respectively

$$\left. \begin{aligned} \delta_1 &= -\frac{b}{10} \sin \frac{\pi y}{b}, & \delta_2 &= \frac{b}{10} \cos \frac{\pi y}{b}, & \delta_3 &= \frac{b \cosh 0.24(h+z)}{10 \cosh 0.24h}, \\ \delta_4 &= -\frac{b \sin \frac{\pi y}{b} \cosh 0.24(h+z)}{10 \cosh 0.24h}, & \delta_5 &= \frac{b \cos \frac{\pi y}{b} \cosh 0.24(h+z)}{10 \cosh 0.24h}. \end{aligned} \right\} \quad (5.2)$$

These continuous shapes are represented in figure 2. Note that each shape is smooth and does not present irregularities that can increase significantly design costs or complicate feasibility of surging WECs in real operational conditions. Note also that configurations 1–2 depend on y only, configuration 3 varies with the vertical z while configurations 4–5 depend on both coordinates. The number 0.24 (in m^{-1}) inside the expressions for $\delta_{3,4,5}$ denotes the wavenumber k_0 corresponding to the eigenfrequency $\omega = 1.4 \text{ rad s}^{-1}$. The latter value has been chosen to check whether the gates interact constructively or destructively in correspondence of that frequency. Now we investigate the effects of both synchronous and subharmonic resonance in terms of power extraction.

5.1. Nonlinear synchronous resonance

Let us consider the evolution equation (4.7). The values of the real coefficients c_L , c_N , c_R , c_S , c_U , c_A and c_B with respect to the eigenfrequency ω are shown in figure 3. The

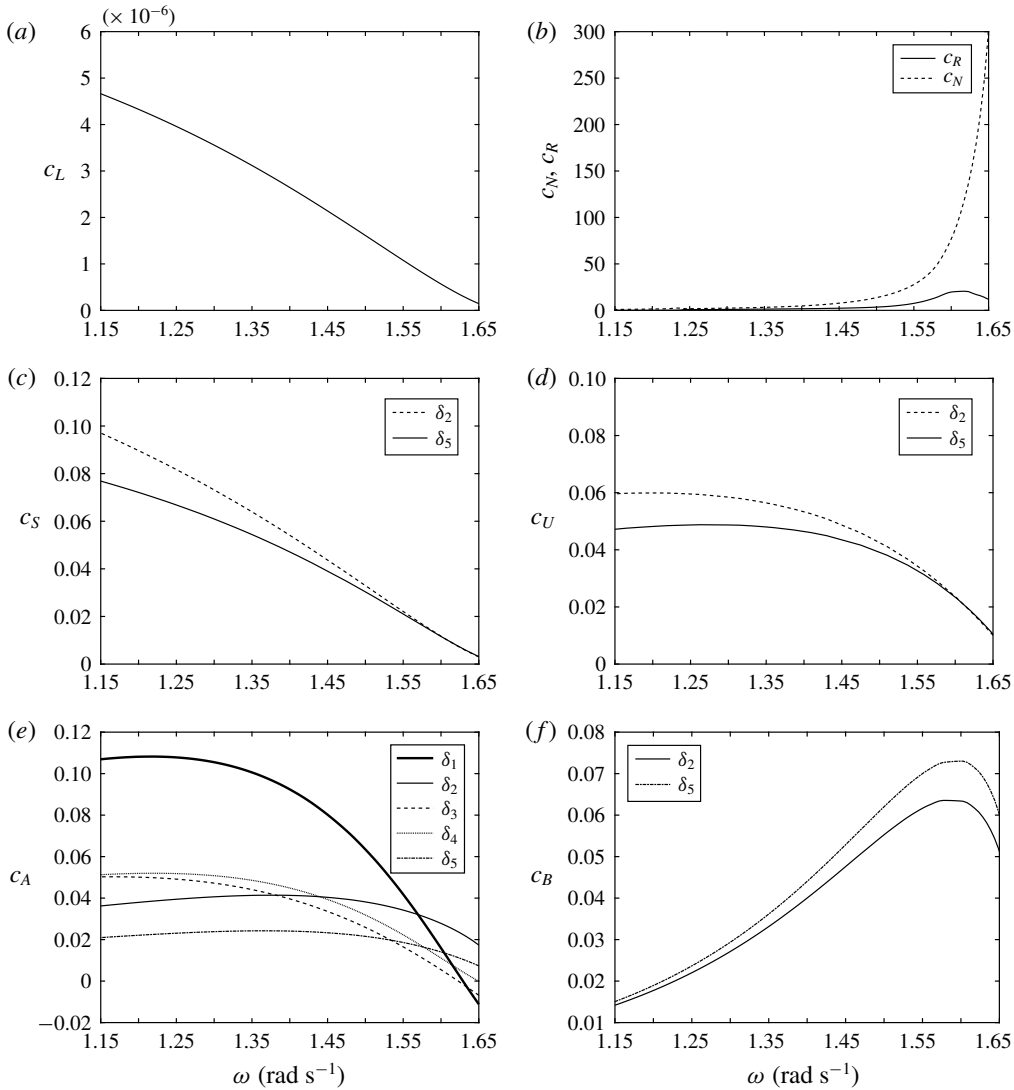


FIGURE 3. Behaviour of the coefficients of the evolution equation (4.7) versus the eigenfrequency ω .

limiting values $\omega = 1.15$ rad s⁻¹ and $\omega = 1.65$ rad s⁻¹ correspond respectively to $c_R = 0$ and the cutoff frequency at which $k_0 = \pi/b$. We remark that c_R must be greater than zero because it represents damping of radiating waves (Li & Mei 2006). Figure 3(e,f) shows that c_A and c_B both have smooth trends without peaks or minima around $\omega = 1.4$ rad s⁻¹. Therefore, contrary to intuition, matching the gate profile to the vertical eigenfunction does not give significant contributions. Sample values of the new forcing coefficients c_S and c_U are shown in figure 3(c,d). Symmetrical configurations about $y = b/2$, i.e. $\delta_{1,3,4}$, yield $c_S = c_U = 0$ and cannot be resonated synchronously.

Let us focus our attention on a fixed eigenfrequency and analyse the effects of the PTO damping coefficient $\nu = \nu_{pto}$ on the dynamic behaviour. For example, take $\omega = 1.2$ rad s⁻¹ and assume two values of ν_{pto} , respectively 10^3 and 10^4 kg s⁻¹.

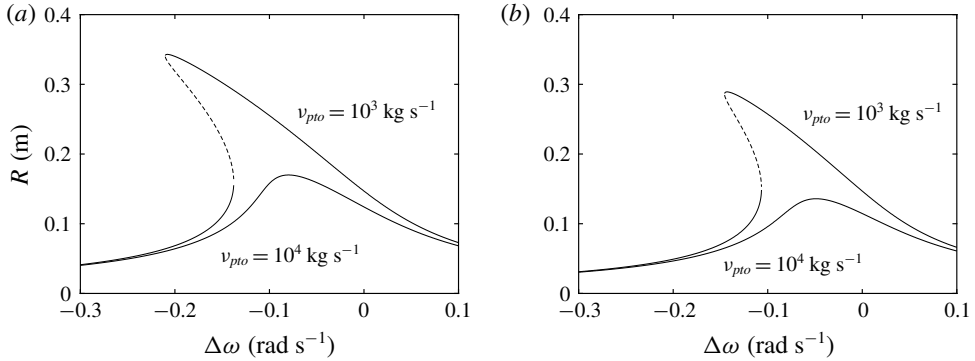


FIGURE 4. Equilibrium branches given by (4.13) versus detuning of the incident wave $\Delta\omega$ for two different values of damping coefficient. (a) Corresponds to the shape function δ_2 while (b) corresponds to δ_5 . The solid lines represent the stable equilibrium branches while the dotted line represents the unstable branch. Trivial fixed points are not possible for synchronous excitation and thus the modal amplitude at the equilibrium is always positive.

The corresponding equilibrium branches defined by (4.13) for configurations $\delta_{2,5}$ are plotted in figure 4. The continuous lines correspond to stable fixed points, while the dashed line is related to unstable saddles (Jordan & Smith 2011). Note that for large values of v_{pto} the unstable fixed point disappears, thus we have one stable fixed point for the entire range of detuning $\Delta\omega$ and absence of non-trivial instability.

Now, we evaluate the efficiency of the system excited through nonlinear synchronous interactions. The generated power by the array is given by

$$P_{sync} = 2v_{pto}(\omega + \Delta\omega)^2 \sum_{q=1}^Q r_q^2 R^2, \quad (5.3)$$

thus, the capture factor (Renzi & Dias 2012) for synchronous resonance is

$$C_{sync}^F = \frac{P_{sync}}{EC_g b}, \quad (5.4)$$

where the term at the denominator represents the incident wave energy flux per array width b (Michele *et al.* 2016b), while

$$EC_g = \frac{\rho g A^2 (\omega + \Delta\omega)}{4k} \left(1 + \frac{2kh}{\sinh 2kh} \right). \quad (5.5)$$

In the latter equation, both group celerity C_g and wavenumber k are related to the frequency $(\omega + \Delta\omega)$. Figure 5 shows the maximum of C_{sync}^F for the configurations $\delta_{2,5}$ versus the eigenfrequency ω for different values of PTO coefficients. The maximum value is ~ 0.7 for δ_2 . This means that even in the presence of small-amplitude incident waves and trapped modes, a device designed to resonate synchronously can still achieve significant efficiency. Note that configuration δ_2 is more efficient than δ_5 . This can be seen from the figures 3(c) and 3(d) representing the behaviour of the forcing coefficients c_S and c_U , respectively. Both c_S and c_U for the second configuration are always greater than the coefficients for configuration 5, hence forcing contributions over δ_2 are greater as well.

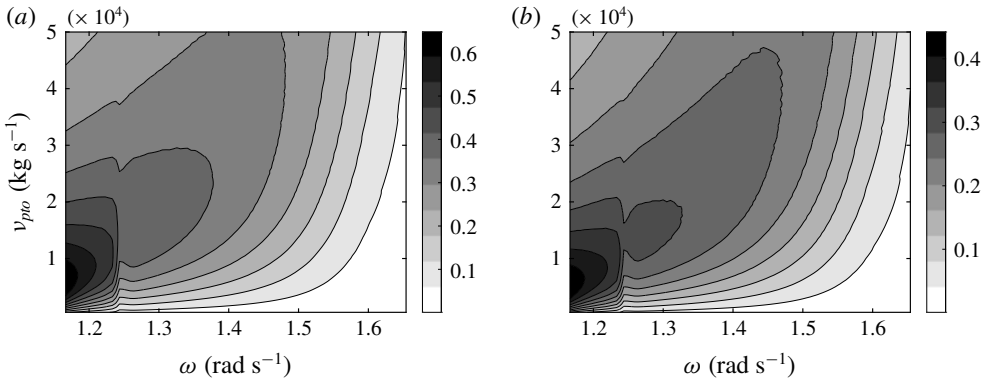


FIGURE 5. Behaviour of the maximum value of the capture factor \mathcal{C}_{sync}^F (5.4) due to nonlinear synchronous resonance. (a) Corresponds to the shape function δ_2 while (b) corresponds to δ_5 .

5.2. Nonlinear subharmonic resonance

In this section we analyse the subharmonic resonance of a single trapped natural mode of an array of Q gates. Subharmonic resonance mechanisms are already known to occur for the Venice gates (Blondeaux, Seminara & Vittori 1993a,b; Mei *et al.* 1994; Vittori *et al.* 1996; Sammarco *et al.* 1997b,a), edge waves (Guza & Bowen 1976; Rockliff 1978; Li 2007), cross-waves (Lichter & Chen 1987), Faraday waves (Miles 1984; Holmes 1986; Gu & Sethna 1987; Miles & Henderson 1990) and trapped waves near a vertical cylinder in a channel (Li & Mei 2006). A similar resonance mechanism has recently been found in the context of WECs by Michele *et al.* (2018b), but for flat gates. Here we explore the role of gate surface curvature in triggering subharmonic resonance effects. Again, the incident waves are assumed small and comparable with $O(\epsilon)$ terms.

Let us assume a small detuning $2\Delta\omega$ between the trapped mode and the incident waves such that $\Delta\omega/\omega = \omega_2\epsilon^2$ (Sammarco *et al.* 1997b). After some lengthy algebra, the evolution equation for subharmonic resonance is

$$-i\vartheta_{t_2} = \vartheta(c_A + ic_B) + \vartheta^2\vartheta^*(c_N + ic_R) + Ae^{-2i(\omega_2 t_2 + (c_8 t_1/\epsilon))}\vartheta^*(c_F + ic_T) + i\vartheta v c_L, \quad (5.6)$$

in which the behaviour of the real forcing coefficients c_F , c_T is represented in figure 6(a). Through the transformation

$$\vartheta = \bar{\vartheta} e^{-i(\omega_2 t_2 + (t_1 c_8/\omega\epsilon))}, \quad (5.7)$$

equation (5.6) becomes

$$-i\bar{\vartheta}_t = \bar{\vartheta}(\Delta\omega + c_A + ic_B) + \bar{\vartheta}^2\bar{\vartheta}^*(c_N + ic_R) + A\bar{\vartheta}^*(c_F + ic_T) + i\bar{\vartheta} v c_L. \quad (5.8)$$

Equation (5.8) is similar to (4.10) and admits both trivial and non-trivial fixed points. For a detailed analysis about its properties we refer to the works of Sammarco *et al.* (1997b) and Michele *et al.* (2018b).

By making use of the polar coordinates R and ψ (or action-angle variables) expressed by $\bar{\vartheta} = i\sqrt{R}e^{i\psi}$, we obtain the two non-trivial fixed points with $R \neq 0$:

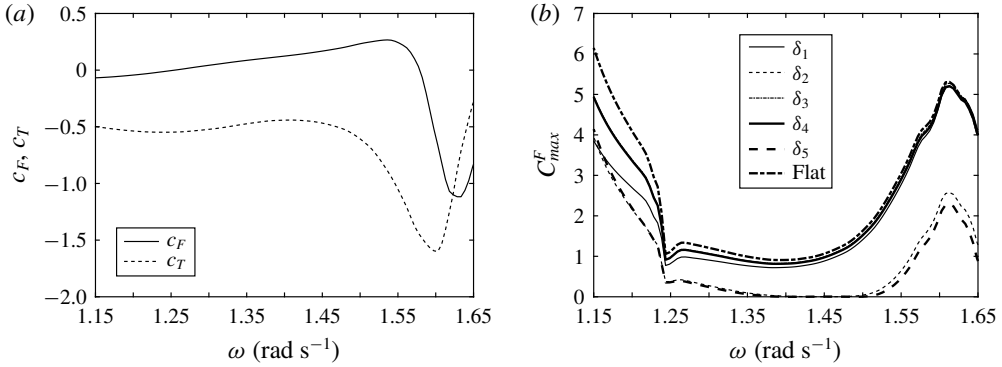


FIGURE 6. (a) Behaviour of the forcing coefficients c_F and c_T and (b) maximum of the optimal capture factor C_{max}^F versus the eigenfrequency of the array ω ; C_{max}^F reaches values greater than 1, i.e. the maximum that can be reached with linear synchronous motion only by an asymmetric WEC.

$$R^\pm = \frac{1}{c_N^2 + c_R^2} \left\{ -c_R(\nu c_L + c_B) - c_N(\Delta\omega + c_A) \pm \sqrt{A^2(c_F^2 + c_T^2)(c_N^2 + c_R^2) - [c_N(\nu c_L + c_B) - c_R(\Delta\omega + c_A)]^2} \right\}. \quad (5.9)$$

Once the stable branch R^+ is evaluated (see Michele *et al.* 2018b), the average generated power by the array due to subharmonic resonance of the natural mode is given by

$$P_{sub} = 2\nu(\omega + \Delta\omega)^2 \sum_{q=1}^Q r_q^2 R^+. \quad (5.10)$$

To find the maximum value of P_{sub} we should equate to zero the partial derivatives of (5.10) with respect to $\Delta\omega$ and ν and then try to seek the roots of the corresponding system. However, finding an analytical expression is not possible and numerical methods are necessary. An optimum criterion can be still defined by approximating the detuning which maximises the generated power with $\Delta\omega = \Delta\omega_{max}$ when the gate oscillation reaches its maximum. To do so, we equate to zero the derivative of (5.10) with respect to ν evaluated at $\Delta\omega_{max}$:

$$\left. \frac{dP_{sub}}{d\nu} \right|_{\Delta\omega=\Delta\omega_{max}} = 0. \quad (5.11)$$

Solution of the latter equation yields

$$\nu = \frac{1}{8c_N c_L} \left\{ 5Ac_N \sqrt{c_F^2 + c_T^2} - 5c_B c_N - 2c_R \omega + \left[9A^2 c_N^2 (c_F^2 + c_T^2) + 9c_B^2 c_N^2 - 4Ac_N c_R \omega \sqrt{c_F^2 + c_T^2} + 4c_R^2 \omega^2 + 2c_B c_N \left(-9Ac_N \sqrt{c_F^2 + c_T^2} + 2c_R \omega \right) \right]^{1/2} \right\}. \quad (5.12)$$

The capture factor for subharmonic resonance \mathcal{C}_{sub}^F becomes

$$\mathcal{C}_{sub}^F = \frac{P_{sub}}{EC_g b}, \quad (5.13)$$

where the term at the denominator is related to the frequency $2(\omega + \Delta\omega)$.

We now define the optimal capture factor \mathcal{C}_{max}^F as the capture factor that satisfies the optimum criterion expressed by (5.11). Figure 6(b) shows the behaviour of \mathcal{C}_{max}^F for each configuration versus the eigenfrequency ω . We obtain that the flat configuration is the most efficient. This is because the coefficient c_B is positive for curved anti-symmetrical shapes and generates hydrodynamic damping, which decreases efficiency. The same figure shows that the optimal capture factor is greater than 1 for a wide range of frequencies. This value corresponds to the maximum efficiency that a two-dimensional asymmetrical wave absorber can reach when excited synchronously in a linearised framework (Mei *et al.* 2005). Thus, nonlinear subharmonic resonance can have beneficial effects on power extraction. Such a result confirms the previous findings of Michele *et al.* (2018b) for flap-type WECs. However, figure 6(b) also reveals that all the curved configurations analysed here are sub-optimal with respect to a flat gate. The occurrence of the shape-dependent term c_B in (5.6) has always resulted to be detrimental to the performance of the system in the case of subharmonic resonance. Therefore, our results suggest that the gate surface curvature does not increase wave power extraction when subharmonic resonance is triggered. This behaviour contrasts the positive impact that using a gate shape has in nonlinear synchronous resonance and in non-resonant states (Hodge *et al.* 2017; Michele *et al.* 2018a).

6. Conclusions

We analysed the hydrodynamic interactions between an array of curved gates and weakly nonlinear waves in a semi-infinite channel. Perturbation expansion and three timing with two slow time scales allowed us to find the complex evolution equations of the Ginzburg–Landau type both for synchronous and subharmonic excitation. New damping, detuning and forcing coefficients that are dependent on the array shape function and its derivatives appear in the equation. We remark also that this dynamics is possible only for nonlinear theories, because forcing contributions now depend on the products between the gate shape function and second-order velocity potentials at the third order.

Then we have applied the theory to the case of surge-type WECs and investigated the effects of curved shapes on the synchronous resonance mechanism by comparing a flat configuration with several curved shapes of practical engineering interest. We also demonstrated that the damping term at the third order is physically coherent with systems for power absorption. We found that effects of synchronous interactions on the generated power can be substantial for optimisation purposes. This highlights the importance of including nonlinear resonances in the cost–benefit analysis when choosing the gate shape.

We have also investigated the case of subharmonic excitation of the WEC array by normally incident waves. In all the cases analysed here, we found that the flat configuration is the most efficient for a wide range of frequencies. Several authors have recently advocated the use of curved gates to improve wave power production in non-resonant states. On the contrary, our results show that curved surfaces are usually penalised if subharmonic resonance occurs.

Finally, we point out that hydrodynamic effects such as fluid shear stresses and vortex shedding in the gap area between adjacent gates are inevitable in real conditions. These damping phenomena contribute to reduce the gate amplitude and should be considered to better evaluate the gate response and the corresponding hydrodynamics.

Acknowledgements

The work of S.M. and E.R. is supported by a Royal Society – CNR International Fellowship. Fruitful discussions with Professors B. Molin, M. Kashiwagi, P. Sclavounos and R. E. Taylor are kindly acknowledged.

Appendix A. Forcing terms and solutions of the boundary-value problems

In this appendix we derive the governing equations and the corresponding solution of each boundary-value problem with order n and harmonic m defined in § 3.

A.1. Leading-order problem $O(1)$ – zeroth harmonic

The forcing terms are all null:

$$\mathcal{F}_{10} = \mathcal{B}_{10} = \mathcal{G}_{10} = \mathcal{D}_{10} = 0. \quad (\text{A } 1)$$

The bounded solution is therefore independent on the fast coordinates, i.e.

$$\phi_{10} = \phi_{10}(t_1, t_2), \quad \chi_{10} = 0, \quad \eta_{10} = 0. \quad (\text{A } 2a-c)$$

A.2. Leading-order problem $O(1)$ – first harmonic and trapped-mode solution

This problem is identical to that in Michele *et al.* (2018b). Hereafter we recall the expression of the leading-order velocity potential

$$\phi_{11} = i\chi\omega \sum_{q=1}^Q \sum_{m=1}^{\infty} \sum_{n=0}^{\infty} \frac{b_{mq}D_n}{C_n\alpha_{nm}} e^{-\alpha_{nm}x} \cos \frac{m\pi y}{b} \cosh k_n(h+z) \equiv i\chi f_{11}(x, y, z), \quad (\text{A } 3)$$

In the latter, the terms k_n are the roots of the dispersion relation

$$\left. \begin{aligned} \omega^2 &= gk_0 \tanh k_0 h, \\ \omega^2 &= -g\bar{k}_n \tan \bar{k}_n h, \quad k_n = i\bar{k}_n, \quad n = 1, \dots, \infty. \end{aligned} \right\} \quad (\text{A } 4)$$

The real coefficients in the expression for the velocity potential ϕ_{11} (A 3) are obtained by matching the velocity of the gate and of the surrounding fluid via (3.27):

$$b_{mq} = r_q \frac{2}{m\pi} \left[\sin \frac{qm\pi}{Q} - \sin \frac{(q-1)m\pi}{Q} \right], \quad (\text{A } 5)$$

$$\alpha_{nm} = \sqrt{\left(\frac{m\pi}{b}\right)^2 - k_n^2}, \quad C_n = \frac{1}{2} \left(h + \frac{g}{\omega^2} \sinh^2 k_n h \right), \quad D_n = \frac{\sinh k_n h}{k_n}. \quad (\text{A } 6a-c)$$

Finally, numerical solution of the equation of motion gives $(Q-1)$ trapped modes and related eigenfrequencies (Sammarco *et al.* 2013).

A.3. Second-order problem $O(\epsilon)$ – zeroth-harmonic drift

Combination of quadratic nonlinearities results in forcing terms of the drift flow:

$$\mathcal{F}_{20} = 0, \quad (\text{A } 7)$$

$$\mathcal{B}_{20} = -\frac{i}{g\epsilon} [\phi_{10,t_1} \epsilon \omega + |\chi|^2 (f_{11,x}^2 + f_{11,y}^2 - f_{11,z}^2)], \quad (\text{A } 8)$$

$$\mathcal{G}_{20} = 0, \quad (\text{A } 9)$$

$$\begin{aligned} \mathcal{D}_{20} = & \frac{\rho}{\epsilon} \int_{(q-1)a}^{qa} dy \left\{ -\frac{\omega^2 |\chi|^2}{g} f_{11}^2 \Big|_{z=0} \right. \\ & \left. + \int_{-h}^0 dz [\phi_{10,t_1} \epsilon \omega + |\chi|^2 (-f_{11,x}^2 + f_{11,y}^2 + f_{11,z}^2)] \right\}_{x=0}. \end{aligned} \quad (\text{A } 10)$$

Since the forcing terms on the free surface \mathcal{F}_{20} and on the gates \mathcal{G}_{20} vanish, the velocity potential ϕ_{20} depends on the slow time scales in the same way as ϕ_{10} :

$$\phi_{20} = \phi_{20}(t_1, t_2). \quad (\text{A } 11)$$

On the other hand, the trapped waves induce a bound wave η_{20} and a static displacement $\chi_{q,20}$ with expressions

$$\eta_{20} = -\frac{i}{g\epsilon} [\phi_{10,t_1} \epsilon \omega + |\chi|^2 (f_{11,x}^2 + f_{11,y}^2 - f_{11,z}^2)], \quad (\text{A } 12)$$

$$\begin{aligned} \chi_{q,20} = & \frac{\rho}{C\epsilon} \int_{(q-1)a}^{qa} dy \left\{ -\frac{\omega^2 |\chi|^2}{g} f_{11}^2 \Big|_{z=0} \right. \\ & \left. + \int_{-h}^0 dz [\phi_{10,t_1} \epsilon \omega + |\chi|^2 (-f_{11,x}^2 + f_{11,y}^2 + f_{11,z}^2)] \right\}_{x=0}, \end{aligned} \quad (\text{A } 13)$$

in which the term ϕ_{10,t_1} is currently unknown and will be obtained at the third order (§ A.7).

A.4. Second-order problem $O(\epsilon)$ – first harmonic and gate shape effects

At the second order, the effects of the gate shape on the total wave field influence the first harmonic. Indeed, the forcing terms are non-null:

$$\mathcal{F}_{21} = -\chi_{t_1} \frac{2\omega^2 f_{11}}{g}, \quad (\text{A } 14)$$

$$\mathcal{B}_{21} = -\chi_{t_1} \frac{i\omega f_{11}}{g}, \quad (\text{A } 15)$$

$$\mathcal{G}_{21} = r\chi_{t_1} \omega + \frac{i\chi}{\epsilon} (f_{11,y} \delta_y + f_{11,z} \delta_z - f_{11,x} \delta_x), \quad (\text{A } 16)$$

$$\mathcal{D}_{21} = 2i\omega^2 M r_q \chi_{t_1} + \frac{\rho}{\epsilon} \int_{(q-1)a}^{qa} dy \int_{-h}^0 dz [i f_{11} \chi_{t_1} \omega \epsilon + \omega f_{11,x} \delta \chi]_{x=0}. \quad (\text{A } 17)$$

Linearity allows us to decompose the velocity potential, i.e. $\phi_{21} = \phi_{21}^G + \phi_{21}^F$, in which ϕ_{21}^F represents the solution with homogenous conditions on the gate, while ϕ_{21}^G is

the solution with homogeneous conditions on the free surface. Both solutions can be found with the eigenfunction expansion method. After some lengthy algebra, we obtain, respectively,

$$\phi_{21}^G = - \sum_{p=0}^{\infty} \sum_{l=0}^{\infty} \cosh k_l(h+z) \cos\left(\frac{p\pi y}{b}\right) \frac{i e^{i\bar{\alpha}_{pl}x}}{\bar{\alpha}_{pl}\epsilon} \left(\sum_{q=1}^Q \frac{\chi_{t1} \omega \epsilon_p b_{pq} D_l \epsilon}{C_l} + i \chi \Delta_{pl} \right), \quad (\text{A } 18)$$

$$\begin{aligned} \phi_{21}^F &= \sum_{p=1}^{\infty} \sum_{l=0}^{\infty} \cosh k_l(h+z) \cos\left(\frac{p\pi y}{b}\right) \frac{2\chi_{t1} \omega^3 \cosh k_l h}{C_l g \alpha_{pl}} \\ &\times \sum_{q=1}^Q \sum_{n=0}^{\infty} \frac{b_{pq} D_n \cosh k_n h (\bar{\alpha}_{pn} e^{-\chi \alpha_{pl}} - \alpha_{pl} e^{-\chi \bar{\alpha}_{pn}})}{\alpha_{pn} C_n (\alpha_{pl}^2 - \bar{\alpha}_{pn}^2)}, \end{aligned} \quad (\text{A } 19)$$

where $\epsilon_0 = 0$ and $\epsilon_p = 1$, $p = 1, \dots$, while the eigenvalue

$$\bar{\alpha}_{pl} = \sqrt{k_l^2 - \left(\frac{p\pi}{b}\right)^2}. \quad (\text{A } 20)$$

The remaining term Δ_{pl} is given by

$$\Delta_{pl} = \frac{1}{C_l \delta_p} \int_0^b dy \int_{-h}^0 \cosh k_l(h+z) \cos\left(\frac{p\pi y}{b}\right) (-f_{11x} \delta + f_{11y} \delta_y + f_{11z} \delta_z) dz, \quad (\text{A } 21)$$

with $\delta_0 = b$ and $\delta_p = b/2$, $p = 1, \dots$.

Note that the potential ϕ_{21} is a peculiar feature of the curved geometry and does not exist for a flat array. Moreover, ϕ_{21}^G (A 18) includes propagating long-crested waves ($p=l=0$) if the gate shape is not symmetric with respect to $y=b/2$. Indeed, such long-crested component is strongly related to the gate shape and would be absent for flat configurations.

A.5. Second-order problem $O(\epsilon)$ – incident waves

The incident wave field is assumed to be at $O(\epsilon)$, hence the incident wave amplitude A' and frequency ω must be an order of magnitude smaller than A'_T , i.e. $A'/A'_T = O(\epsilon)$. The corresponding velocity potential is given by

$$\phi^I = - \frac{iAg}{2\epsilon\omega} \frac{\cosh k_0(h+z)}{\cosh k_0 h} e^{-ik_0 x}, \quad (\text{A } 22)$$

the scattering potential is

$$\phi^S = - \frac{iAg}{2\epsilon\omega} \frac{\cosh k_0(h+z)}{\cosh k_0 h} e^{ik_0 x}, \quad (\text{A } 23)$$

while the radiation potential due to the in-phase motion of the gates is given by

$$\phi^R = - \sum_{l=0}^{\infty} \frac{\omega \chi^A D_n}{k_n C_n} \cosh k_n(h+z) e^{ik_n x}. \quad (\text{A } 24)$$

The response χ^A is

$$\chi^A = \frac{-\rho a A g D_0 / (\epsilon \cosh k_0 h)}{-\omega^2 M + C - i\omega^2 \rho a \sum_{l=0}^{\infty} \frac{D_n^2}{k_n C_n}}, \quad (\text{A } 25)$$

where the wavenumbers k_n correspond the real roots of the dispersion relation (A 4).

A.6. Second-order problem $O(\epsilon)$ – radiated second harmonic

The second harmonic is forced both on the free surface and on the gate and does not depend on the shape of the gate. The forcing terms are

$$\mathcal{F}_{22} = -\frac{1}{\epsilon g} [\eta_{11}(-\omega^2 \phi_{11z} + g \phi_{11zz}) - 2i\omega |\nabla \phi_{11}|^2], \quad (\text{A } 26)$$

$$\mathcal{B}_{22} = -\frac{1}{\epsilon g} \left[-i\omega \phi_{11z} \eta_{11} + \frac{|\nabla \phi_{11}|^2}{2} \right], \quad (\text{A } 27)$$

$$\mathcal{G}_{22} = -\frac{\phi_{11xx} \chi_{11}}{\epsilon}, \quad (\text{A } 28)$$

$$\mathcal{D}_{22} = \frac{\rho}{\epsilon} \int_0^b dy \left\{ \frac{g \eta_{11}^2}{2} - i\omega \phi_{11} \eta_{11}|_{z=0} + \int_{-h}^0 \left(-i\omega \phi_{11x} r \chi + \frac{|\nabla \phi_{11}|^2}{2} \right) dz \right\}_{x=0}. \quad (\text{A } 29)$$

Similarly to the case for the first-harmonic solution at $O(\epsilon)$, we decompose the velocity potential as $\phi_{22}^* = \phi_{22}^G + \phi_{22}^F$, where ϕ_{22}^F represents the solution with homogenous conditions on the gate, while ϕ_{22}^G is the solution with homogeneous conditions on the free surface. Again, the solution for both potentials can be obtained in terms of vertical and horizontal eigenfunction expansions:

$$\begin{aligned} \phi_{22}^G = & -\chi^2 \omega \sum_{p=0}^{\infty} \sum_{l=0}^{\infty} \cosh \kappa_l(h+z) \cos\left(\frac{p\pi y}{b}\right) \frac{e^{i\hat{\alpha}_{pl}x}}{\hat{\alpha}_{pl}} \left[\sum_{q=1}^{\infty} \frac{2id_{pq}F_l}{E_l} \right. \\ & \left. + \frac{\delta_p}{b\epsilon} \sum_{q=1}^Q \sum_{r=1}^Q \sum_{n=0}^{\infty} \sum_{m=-\infty}^{\infty} \frac{b_{mq}b_{(m+p)r}\alpha_{nm}D_nD_{nl}}{E_lC_n} \right], \quad m \neq 0, m+p \neq 0, \quad (\text{A } 30) \end{aligned}$$

$$\begin{aligned} \phi_{22}^F = & \frac{i\chi^2\omega}{4bg\epsilon} \sum_{p=0}^{\infty} \sum_{l=0}^{\infty} \sum_{q=1}^Q \sum_{r=1}^Q \sum_{n=0}^{\infty} \sum_{s=0}^{\infty} \sum_{m=-\infty}^{\infty} \cosh \kappa_l(h+z) \cos\left(\frac{p\pi y}{b}\right) \cosh k_n h \cosh k_s h \\ & \times \frac{\delta_p \beta_{qnm} \beta_{r(m+p)s} \cosh \kappa_l h [\hat{\alpha}_{pl} e^{-x(\alpha_{nm} + \alpha_{(m+p)s})} - i(\alpha_{nm} + \alpha_{(m+p)s}) e^{i\hat{\alpha}_{pl}x}]}{E_l [\hat{\alpha}_{pl}^2 + (\alpha_{nm} + \alpha_{(m+p)s})^2]} \\ & \times \left(\frac{3\omega^4}{g^2} - k_n^2 + \frac{2m\pi(m+p)}{b^2} + 2\alpha_{nm}\alpha_{(m+p)s} \right), \quad m \neq 0, m+p \neq 0, \quad (\text{A } 31) \end{aligned}$$

where

$$D_{nl} = \frac{\kappa_l \cosh k_n h \sinh \kappa_l h - k_n \cosh \kappa_l h \sinh k_n h}{\kappa_l^2 - k_n^2}, \quad (\text{A } 32)$$

$$\beta_{qnm} = \frac{\omega b_{mq} D_n}{\alpha_{nm} C_n}, \quad \hat{\alpha}_{pl} = \sqrt{\kappa_l^2 - \left(\frac{p\pi}{b}\right)^2}. \quad (\text{A } 33a, b)$$

The term d_{pq} corresponds to the Fourier coefficient for the gate displacement:

$$d_{pq} = \chi_{q,22} \frac{2}{p\pi} \left[\sin \frac{qp\pi}{Q} - \sin \frac{(q-1)p\pi}{Q} \right], \quad (\text{A } 34)$$

in which $\chi_{q,22}$ denotes the displacement of the q th gate normalised with respect to $i\chi^2$. Unlike the case of Sammarco *et al.* (1997b), the forcing term \mathcal{D}_{22} given by (A 29)

does not vanish, hence the displacement related to the second harmonic differs from zero. Indeed, usage of the equation of motion (3.28) gives a forced linear system for the unknowns $\chi_{q,22}$, $q = 1, \dots, Q$. The corresponding determinant of the coefficient matrix differs from zero, thus the inhomogeneous solution is unique. After some algebra we obtain

$$\begin{aligned} & \chi_{j,22}(-4\omega^2 M + C) - 4i\omega^2 \rho \sum_{q=1}^Q \sum_{p=0}^{\infty} \sum_{l=0}^{\infty} \frac{d_{pq} c_{pj} F_l^2}{\hat{\alpha}_{pl} E_l} \\ &= \frac{\rho}{\epsilon} \left\{ -\frac{i\omega\delta_p}{2b} \sum_{p=0}^{\infty} \sum_{l=0}^{\infty} \sum_{q=1}^Q \sum_{r=1}^Q \sum_{n=0}^{\infty} \sum_{s=0}^{\infty} \sum_{m=-\infty}^{\infty} \right. \\ & \quad \times \left(\frac{3\omega^4}{g^2} - k_n^2 + \frac{2m\pi(m+p)}{b^2} + 2\alpha_{nm}\alpha_{(m+p)s} \right) \\ & \quad \times \frac{\omega F_l c_{pj} \beta_{qnm} \beta_{r(m+p)s} \cosh \kappa_l h [\hat{\alpha}_{pl} - i(\alpha_{nm} + \alpha_{(m+p)s})]}{g \hat{\alpha}_{pl} E_l [\hat{\alpha}_{pl}^2 + (\alpha_{nm} + \alpha_{(m+p)s})^2]} \\ & \quad + \sum_{p=0}^{\infty} \sum_{l=0}^{\infty} \sum_{q=1}^Q \sum_{r=1}^Q \sum_{n=0}^{\infty} \sum_{m=-\infty}^{\infty} \frac{i\omega c_{pj} b_{mq} b_{(m+p)r} \alpha_{nm} F_l D_n D_{nl}}{\alpha_{nm} \hat{\alpha}_{pl} E_l C_n} \\ & \quad \left. + i \left(\frac{3ah\omega^2 r_j^2}{2} + \int_{(q-1)a}^{qa} dy \int_{-h}^0 \frac{f_{11z}^2 + f_{11y}^2}{2} dz + \int_{(q-1)a}^{qa} \frac{\omega^2 f_{11}^2}{2g} dy \right) \right\}, \quad (\text{A } 35) \end{aligned}$$

where the term c_{pj} is given by

$$c_{pj} = \frac{b}{p\pi} \left[\sin \frac{jp\pi}{Q} - \sin \frac{(j-1)p\pi}{Q} \right]. \quad (\text{A } 36)$$

A.7. The third-order problem $O(\epsilon^2)$ – zeroth harmonic

For the zeroth harmonic at $O(\epsilon^2)$ the forcing terms respectively on the free surface and on the gate are

$$\begin{aligned} \mathcal{F}_{30} = & -\omega^2 \phi_{10,t_1} + \frac{1}{g\epsilon} \left\{ \frac{3\omega^4 f_{11} \omega (\chi \chi_{t_1}^* + \chi^* \chi_{t_1})}{g} + \omega f_{11} (\chi^* \phi_{21,zz} + \chi \phi_{21,zz}^*) \right. \\ & \left. - 2\omega |\chi|_{t_1}^2 |\nabla f_{11}|^2 + \omega f_{11,z} [\chi^* (-\phi_{21} + \chi_{t_1} f_{11}) - \chi (\phi_{21}^* + \chi_{t_1}^* f_{11})] \right\}, \quad (\text{A } 37) \end{aligned}$$

$$\mathcal{G}_{30} = -\frac{r}{\epsilon} (\chi^* \phi_{21,xx} + \chi \phi_{21,xx}^*) + \omega \chi_{20,t_1}. \quad (\text{A } 38)$$

Note that the dependence of ϕ_{10} (A 2) on the slow time coordinate t_1 is still unknown. Application of Green's theorem to ϕ_{10} and ϕ_{30} yields

$$-\int_0^b dy \int_0^X \mathcal{F}_{30}|_{z=0} dx + \int_0^b dy \int_{-h}^0 \mathcal{G}_{30}|_{x=0} dz - \int_0^b dy \int_{-h}^0 \phi_{10} \phi_{30,x}|_{x=X} dz = 0, \quad (\text{A } 39)$$

where $X \gg 1$ corresponds to a large value of the x -coordinate. Solving the integrals and requesting that $|\nabla \phi_{30}| \rightarrow 0$ as $X \rightarrow +\infty$, we obtain

$$\phi_{10,t_1} = 0, \quad (\text{A } 40)$$

in the limit $X \rightarrow +\infty$. Therefore, the bounded non-secular solution of (A 40) is simply $\phi_{10} = 0$.

A.8. The third-order problem $O(\epsilon^2)$ – first harmonic and forcing terms

The forcing terms \mathcal{F}_{31} , \mathcal{G}_{31} and \mathcal{D}_{31} are given by the following expressions:

$$\begin{aligned} \mathcal{F}_{31} = & \frac{\omega^2}{g}(2i\phi_{21t_1} - 2\chi_{t_2} - f_{11}\chi_{t_1t_1}) + \frac{1}{g\epsilon} \left\{ \frac{\omega f_{11}\chi^*}{g}(-4\omega^2\phi_{22z} + g\phi_{22zz}) \right. \\ & + \frac{i\chi^*}{g} \left[-2i\omega\phi_{22} - \frac{\chi^2}{\epsilon} \left(\frac{3f_{11z}^2}{2} + \frac{f_{11x}^2 + f_{11y}^2}{2} \right) \right] (\omega^2 f_{11z} - g f_{11z}) \\ & - \frac{i\omega^2 f_{11}^2 \chi |\chi|^2}{2\epsilon g^2} (-\omega^2 f_{11zz} + g f_{11zzz}) + 2\omega\chi^* (\nabla f_{11} \cdot \nabla \phi_{22}) \\ & + \frac{2i\omega^2 \chi |\chi|^2 f_{11} |\nabla f_{11}|_z^2}{\epsilon g} - \frac{3i\chi |\chi|^2 \nabla f_{11} \cdot \nabla |\nabla f_{11}|^2}{2\epsilon} \\ & \left. - \frac{\chi |\chi|^2}{g\epsilon} (f_{11x}^2 + f_{11y}^2 - f_{11z}^2) (-\omega^2 f_{11z} + g f_{11zz}) \right\}, \end{aligned} \quad (\text{A } 41)$$

$$\begin{aligned} \mathcal{G}_{31} = & r\omega\chi_{t_2} + \frac{1}{\epsilon} \left\{ -\chi |\chi|^2 f_{11xx} \chi_{q,22} - \chi^* r \phi_{22xx} - \phi_{21xx} \delta - \frac{\chi |\chi|^2 f_{11xxx} r}{2\epsilon} \right. \\ & \left. - \frac{i\chi f_{11xxx} \delta^2}{2\epsilon} + \phi_{21y} \delta_y + \phi_{21z} \delta_z + \frac{i\chi \delta}{\epsilon} (f_{11xy} \delta_y + f_{11xz} \delta_z) \right\}, \end{aligned} \quad (\text{A } 42)$$

$$\begin{aligned} \mathcal{D}_{31} = & \frac{\rho}{\epsilon} \int_{(q-1)a}^{qa} dy \int_{-h}^0 dz \left\{ i\epsilon\omega\chi_{t_2} f_{11} + \epsilon\omega\phi_{21t_1} - i\omega\delta\phi_{21x} + i\epsilon\omega\chi_{t_1} f_{11x} \delta - \frac{\omega^2 r \chi \chi_{20}}{\epsilon} \right. \\ & + i\chi |\chi|^2 f_{11x} \chi_{q,22} + \frac{\chi}{2\epsilon} (3|\chi|^2 f_{11xx} \omega r_q^2 + f_{11xx} \omega \delta^2 + |\chi|^2 r_q |\nabla f_{11}|_x^2) \\ & \left. - i\chi^* \nabla f_{11} \cdot \nabla \phi_{22} \right\}_{x=0} + \frac{\rho}{\epsilon} \int_{(q-1)a}^{qa} dy \left\{ \frac{2i\omega^2 f_{11} \phi_{22}}{g} + \frac{3\chi |\chi|^2 f_{11} f_{11}^2 \omega^3}{2g^2 \epsilon} \right. \\ & \left. - \frac{\chi |\chi|^2 f_{11} \omega}{2g\epsilon} (6f_{11x} \omega + |\nabla f_{11}|^2) \right\}_{x=0, z=0} - M\omega^2 (-2i\chi_{t_2} + \chi_{t_1t_1}) + \frac{ir_q \omega v \chi}{\epsilon^2}. \end{aligned} \quad (\text{A } 43)$$

Appendix B. Expressions for the coefficient of the evolution equations (4.10) and (5.8)

Let us decompose ϕ_{21} as follows:

$$\phi_{21} = \chi_{t_1} \frac{\phi_{21}^{(1)}}{\epsilon} + \chi \frac{\phi_{21}^{(2)}}{\epsilon}. \quad (\text{B } 1)$$

The expressions for the synchronous forcing coefficients are

$$c_S = \text{Re} \left\{ \frac{iv^S}{\widehat{v}} \right\}, \quad c_U = \text{Im} \left\{ \frac{iv^S}{\widehat{v}} \right\}, \quad (\text{B } 2a, b)$$

with

$$v^S = \int_0^b dy \int_{-h}^0 \{ f_{11} (\phi_{xx}^A \delta - \phi_z^A \delta_z) + r\omega \delta \phi_x^A \} dz. \quad (\text{B } 3)$$

In the latter, the velocity potential $\phi^A = \phi^I + \phi^S + \phi^R$ refers to the first-harmonic incident, scattered and in-phase radiation potentials (A 22)–(A 24). The expressions for the remaining coefficients inside the evolution equations (4.10)–(5.8) are

$$c_A = \operatorname{Re} \left\{ \frac{i}{\widehat{v}} (\nu + ic_\delta \widetilde{v} - c_\delta^2 \overline{v}) \right\}, \quad c_B = \operatorname{Im} \left\{ \frac{i}{\widehat{v}} (\nu + ic_\delta \widetilde{v} - c_\delta^2 \overline{v}) \right\}, \quad (\text{B } 4a, b)$$

$$c_N = \operatorname{Re} \left\{ \frac{i\nu^R}{\widehat{v}} \right\}, \quad c_R = \operatorname{Im} \left\{ \frac{i\nu^R}{\widehat{v}} \right\}, \quad c_F = \operatorname{Re} \left\{ \frac{i\nu^I}{\widehat{v}} \right\}, \quad c_T = \operatorname{Im} \left\{ \frac{i\nu^I}{\widehat{v}} \right\}, \quad (\text{B } 5a-d)$$

$$c_L = -\frac{i}{\widehat{v}} \sum_{q=1}^Q \frac{\omega r_q^2}{\rho}, \quad (\text{B } 6)$$

where

$$\widehat{v} = -\int_0^b dy \int_0^{+\infty} \frac{2f_{11}^2 \omega}{g} dx - \int_0^b dy \int_{-h}^0 2f_{11} r dz - \sum_{q=1}^Q \frac{2r_q^2 \omega M}{\rho}, \quad (\text{B } 7)$$

$$\begin{aligned} \nu = \int_0^b dy \int_{-h}^0 dz \left\{ -f_{11} \left[-\delta \phi_{21xx}^{(2)} + \delta_y \phi_{21y}^{(2)} + \delta_z \phi_{21z}^{(2)} - \frac{i\delta^2 f_{11xxx}}{2} \right. \right. \\ \left. \left. + i\delta(f_{11xy} \delta_y + f_{11xz} \delta_z) \right] + ir\delta\omega \left(-\frac{\delta f_{11xx}}{2} + f_{11y} \delta_y + f_{11z} \delta_z \right) \right\}, \end{aligned} \quad (\text{B } 8)$$

$$\begin{aligned} \widetilde{v} = \int_0^b dy \int_0^{+\infty} \frac{2if_{11}\phi_{21}^{(2)}\omega}{g} dx + \int_0^b dy \int_{-h}^0 dz \{ -f_{11}[-\delta\phi_{21xx}^{(1)} + \delta_y\phi_{21y}^{(1)} + \delta_z\phi_{21z}^{(1)}] \\ + ir(-2i\delta\omega r + \phi_{21}^{(2)}) \}, \end{aligned} \quad (\text{B } 9)$$

$$\overline{v} = \int_0^b dy \int_0^{+\infty} \frac{2if_{11}\phi_{21}^{(1)}\omega - if_{11}^2}{g} dx + \int_0^b dy \int_{-h}^0 ir\phi_{21}^{(1)} dz + \sum_{q=1}^Q \frac{iMr_q^2}{\rho}, \quad (\text{B } 10)$$

$$\begin{aligned} \nu^I = \int_0^b dy \int_0^{+\infty} f_{11} \left\{ \frac{f_{11}\omega}{g} (-4\omega^2 \tilde{\phi}_z^A + g\tilde{\phi}_{zz}^A) + \frac{2\omega\tilde{\phi}^A}{g} (\omega^2 f_{11z} - g f_{11zz}) \right. \\ \left. + 2\omega \nabla f_{11} \cdot \nabla \tilde{\phi}^A \right\} dx + \int_0^b dy \int_{-h}^0 dz \{ -f_{11} (i\tilde{\chi}^A f_{11x} - r_Q \tilde{\phi}_{xx}^A) \\ + ir(-2i\omega r \tilde{\phi}_x^A + \omega f_{11x} \tilde{\chi}^A - i\nabla f_{11} \cdot \nabla \tilde{\phi}^A) \} - \int_0^b \frac{2r\omega^2 f_{11} \tilde{\phi}^A}{g} dy, \end{aligned} \quad (\text{B } 11)$$

$$\begin{aligned} \nu^R = \int_0^b dy \int_0^{+\infty} f_{11} \left\{ \frac{\omega f_{11}}{g} (-4\omega^2 \phi_{22z}^\chi + g\phi_{22zz}^\chi) - \frac{i\omega^2 f_{11}^2}{2g^2} (-\omega^2 f_{11zz} + g f_{11zzz}) \right. \\ \left. + \frac{i}{g} \left[-2i\omega \phi_{22}^\chi - \left(\frac{3\omega^4 f_{11}^2}{2g} + \frac{f_{11x}^2 + f_{11y}^2}{2} \right) \right] \left(\frac{\omega^4 f_{11}}{g} - g f_{11zz} \right) + 2\omega \nabla f_{11} \cdot \nabla \phi_{22}^\chi \right. \\ \left. + \frac{2i\omega^2 f_{11} |\nabla f_{11}|_z^2}{g} - \frac{3i\nabla f_{11} \cdot \nabla |\nabla f_{11}|^2}{2} - \frac{1}{g} \left(-\frac{\omega^4 f_{11}}{g^2} + f_{11y}^2 + f_{11z}^2 \right) \right\} \end{aligned}$$

$$\begin{aligned}
& \times \left(-\frac{\omega^4 f_{11}}{g} + g f_{11zz} \right) \Big\} dx + \int_0^b dy \int_{-h}^0 dz \left\{ f_{11} \left(i f_{11xx} \chi_{q,22} - \phi_{22}^\chi r - \frac{i f_{11xxx} r^2}{2} \right) \right. \\
& + i r_q \left(-2i\omega \phi_{22x}^\chi r + \omega \chi_{q,22} f_{11x} - i \nabla f_{11} \cdot \nabla \phi_{22}^\chi + \frac{3\omega f_{11xx} r^2}{2} + \frac{r_q |f_{11x}|^2}{2} \right) \Big\} \\
& + \int_0^b \frac{\omega f_{11}}{g} \left\{ 2i\omega \phi_{22}^\chi + \frac{3\omega^2 f_{11} f_{11z}}{2g} - \frac{|\nabla f_{11}|^2}{2} \right\} dy - \sum_{q=1}^Q i a h \omega^2 r_q^2 \chi_{20}, \quad (\text{B } 12)
\end{aligned}$$

in which the terms $\tilde{\phi}^A$ and $\tilde{\chi}^A$ have the same structure of ϕ^A and χ^A with incident wave frequency 2ω .

REFERENCES

- ADAMO, A. & MEI, C. C. 2005 Linear response of Venice storm gates to incident waves. *Proc. R. Soc. Lond. A* **461**, 1711–1734.
- ALAM, M.-R., LIU, Y. & YUE, D. K.-P. 2010 Oblique sub- and super-harmonic bragg resonance of surface waves by bottom ripples. *J. Fluid Mech.* **643**, 437–447.
- ARANSON, I. & KRAMER, L. 2002 The world of the complex Ginzburg–Landau equation. *Rev. Mod. Phys.* **74** (1), 99–143.
- BABARIT, A. 2018 *Ocean Wave Energy Conversion*. Elsevier.
- BABARIT, A., HALS, J., MULIAWAN, M. J., KURNIAWAN, A., MOAN, T. & KROKSTAD, J. 2012 Numerical benchmarking study of a selection of wave energy converters. *Renew. Energy* **41**, 44–63.
- BLONDEAUX, P., SEMINARA, G. & VITTORI, G. 1993a Linear response of the gate system for protection of the Venice lagoon. Note I. Transverse free modes. *Rend. Mat. Acc. Lincei A* **59**, 291–298.
- BLONDEAUX, P., SEMINARA, G. & VITTORI, G. 1993b Linear response of the gate system for protection of the Venice lagoon. Note II. Excitation of transverse subharmonic modes. *Rend. Mat. Acc. Lincei A* **59**, 299–305.
- BLONDEAUX, P. & VITTORI, G. 1995 The nonlinear excitation of synchronous edge waves by a monochromatic wave normally incident on a plane beach. *J. Fluid Mech.* **301**, 251–268.
- CALLAN, M., LINTON, C. & EVANS, D. V. 1991 Trapped modes in two-dimensional waveguides. *J. Fluid Mech.* **229**, 51–64.
- DIAS, F., RENZI, E., GALLAGHER, S., SARKAR, D., WEI, Y., ABADIE, T., CUMMINS, C. & RAFIEE, A. 2017 Analytical and computational modelling for wave energy systems: the example of oscillating wave surge converters. *Acta Mechanica Sin.* **33**, 647–662.
- DRAZIN, P. G. 2002 *Introduction to Hydrodynamic Stability*. Cambridge University Press.
- EVANS, D. V. & LINTON, C. M. 1991 Trapped modes in open channels. *J. Fluid Mech.* **225**, 153–175.
- EVANS, D. V. & PORTER, R. 1997 Trapped modes about multiple cylinders in a channel. *J. Fluid Mech.* **339**, 331–356.
- FOLLEY, M., WHITTAKER, T. & VAN’T HOFF, J. 2007 The design of small seabed mounted bottom-hinged wave energy converters. In *7th European Wave and Tidal Energy Conference, Porto, PT*.
- GU, X. M. & SETHNA, P. R. 1987 Resonant surface waves and chaotic phenomena. *J. Fluid Mech.* **183**, 543–565.
- GUZA, R. T. & BOWEN, A. J. 1976 Finite amplitude stokes edge waves. *J. Mar. Res.* **34**, 269–293.
- HEIN, S. & KOCH, W. 2008 Acoustic resonances and trapped modes in pipes and tunnels. *J. Fluid Mech.* **605**, 401–428.
- HENRY, A., DOHERTY, K., CAMERON, L., WHITTAKER, T. & DOHERTY, R. 2010 Advances in the design of the oyster wave energy converter. *Marine Renewable and Offshore Wind Energy*. Royal Institution of Naval Architects.

- HODGE, C. W., BATEMAN, W., YUAN, Z., THIES, P. & BRUCE, T. 2017 Performance analysis of the CCell wave energy device. In *12th European Wave and Tidal Energy Conference, Cork, IE*.
- HOLMES, P. 1986 Chaotic motion in a weakly nonlinear model for surface waves. *J. Fluid Mech.* **162**, 365–388.
- JORDAN, D. W. & SMITH, P. 2011 *Nonlinear Ordinary Differential Equations*. Oxford University Press.
- KIRBY, J. 1986 A general wave equation for waves over rippled beds. *J. Fluid Mech.* **162**, 171–186.
- KUZNETSOV, N. 1993 Trapped modes of internal waves in a channel spanned by a submerged cylinder. *J. Fluid Mech.* **254**, 113–126.
- LI, G. 2007 Nonlinear resonance of trapped waves on a plane beach. PhD thesis, Massachusetts Institute of Technology.
- LI, G. & MEI, C. C. 2003 Natural modes of mobile flood gates. *Appl. Ocean Res.* **25**, 115–126.
- LI, Y. & MEI, C. C. 2006 Subharmonic resonance of a trapped wave near a vertical cylinder in a channel. *J. Fluid Mech.* **561**, 391–416.
- LICHTER, S. & CHEN, J. 1987 Subharmonic resonance of nonlinear cross-waves. *J. Fluid Mech.* **183**, 451–465.
- LINTON, C. & MCIVER, P. 2001 *Mathematical Techniques for Wave/Structure Interactions*. Chapman & Hall/CRC.
- LINTON, C. M. & RATCLIFFE, K. 2004 Bound states in coupled guides. I. Two dimensions. *J. Math. Phys.* **45**, 1359–1379.
- MEI, C. C., HARA, T. & NACIRI, M. 1988 Note on Bragg scattering of water waves by parallel bars on the seabed. *J. Fluid Mech.* **186**, 147–162.
- MEI, C. C., SAMMARCO, P., CHAN, E. & PROCACCINI, C. 1994 Subharmonic resonance of proposed storm gates for Venice lagoon. *Proc. R. Soc. Lond. A* **444**, 257–265.
- MEI, C. C., STIASSNIE, M. & YUE, D. K.-P. 2005 *Theory and Application of Ocean Surface Waves*. World Scientific.
- MICHELE, S., RENZI, E. & SAMMARCO, P. 2018a A second-order theory for wave energy converters with curved geometry. In *33rd International Workshop on Water Waves and Floating Bodies, Guidel-Plages, FR*.
- MICHELE, S., SAMMARCO, P. & D'ERRICO, M. 2016a The optimal design of a flap gate array in front of a straight vertical wall: resonance of the natural modes and enhancement of the exciting torque. *Ocean Engng* **118**, 152–164.
- MICHELE, S., SAMMARCO, P. & D'ERRICO, M. 2016b Theory of the synchronous motion of an array of floating flap gates oscillating wave surge converter. *Proc. R. Soc. Lond. A* **472**, 20160174.
- MICHELE, S., SAMMARCO, P. & D'ERRICO, M. 2018b Weakly nonlinear theory for oscillating wave surge converters in a channel. *J. Fluid Mech.* **834**, 55–91.
- MICHELE, S., SAMMARCO, P., D'ERRICO, M., RENZI, E., ABDOLALI, A., BELLOTTI, G. & DIAS, F. 2015 Flap gate farm: from Venice lagoon defense to resonating wave energy production. Part 2. Synchronous response to incident waves in open sea. *Appl. Ocean Res.* **52**, 43–61.
- MILES, J. 1984 Nonlinear Faraday resonance. *J. Fluid Mech.* **146**, 285–302.
- MILES, J. & HENDERSON, D. 1990 Parametrically forced surface waves. *Annu. Rev. Fluid Mech.* **22**, 143–165.
- NAYFEH, A. H. & MOOK, D. T. 1995 *Nonlinear Oscillations*. Wiley Classic Library.
- NAZAROV, S. & VIDEMAN, H. 2009 A sufficient condition for the existence of trapped modes for oblique waves in a two-layer fluid. *Proc. R. Soc. Lond. A* **465**, 3799–3816.
- NOAD, I. & PORTER, R. 2015 Optimisation of arrays of flap-type oscillating wave surge converters. *Appl. Ocean Res.* **50**, 237–253.
- PORTER, R. 2007 Trapped modes in thin elastic plates. *Wave Motion* **45**, 3–15.
- PORTER, R. & EVANS, D. V. 1999 Rayleigh–Bloch surface waves along periodic gratings and their connection with trapped modes in waveguides. *J. Fluid Mech.* **386**, 233–258.
- RENZI, E., ABDOLALI, A., BELLOTTI, G. & DIAS, F. 2014 Wave-power absorption from a finite array of oscillating wave surge converters. *Renew. Energy* **63**, 55–68.

- RENZI, E. & DIAS, F. 2012 Resonant behaviour of an oscillating wave energy converter in a channel. *J. Fluid Mech.* **701**, 482–510.
- RENZI, E. & DIAS, F. 2013 Hydrodynamics of the oscillating wave surge converter in the open ocean. *Eur. J. Mech. (B/Fluids)* **41**, 1–10.
- RENZI, E. & DIAS, F. 2014 Motion resonant modes of large articulated damped oscillators in waves. *J. Fluid Struct.* **49**, 705–715.
- ROCKLIFF, N. 1978 Finite amplitude effects in free and forced edge waves. *Math. Proc. Camb. Phil. Soc.* **83**, 463–479.
- SAMMARCO, P., MICHELE, S. & D'ERRICO, M. 2013 Flap gate farm: from Venice lagoon defense to resonating wave energy production. Part 1. Natural modes. *Appl. Ocean Res.* **43**, 203–213.
- SAMMARCO, P., TRAN, H. H., GOTTLIEB, O. & MEI, C. C. 1997a Subharmonic resonance of Venice gates in waves. Part 2. Sinusoidally modulated incident waves. *J. Fluid Mech.* **349**, 327–359.
- SAMMARCO, P., TRAN, H. H. & MEI, C. C. 1997b Subharmonic resonance of Venice gates in waves. Part 1. Evolution equation and uniform incident waves. *J. Fluid Mech.* **349**, 295–325.
- SARKAR, D., RENZI, E. & DIAS, F. 2015 Effect of a straight coast on the hydrodynamics and performance of the oscillating wave surge converter. *Ocean Engng* **105**, 25–32.
- SCHMITT, P. & ELSAESSER, B. 2015 On the use of OpenFOAM to model oscillating wave surge converters. *Ocean Engng* **108**, 98–104.
- UTSUNOMYA, T. & TAYLOR, R. E. 1997 Trapped modes around a row of circular cylinders in a channel. *J. Fluid Mech.* **339**, 331–356.
- VITTORI, G., BLONDEAUX, P. & SEMINARA, G. 1996 Waves of finite amplitude trapped by oscillating gates. *Proc. R. Soc. Lond. A* **452**, 791–811.
- WILKINSON, L., WHITTAKER, T., THIES, P. R., DAY, S. & INGRAM, D. 2017 The power-capture of a nearshore, modular, flap-type wave energy converter in regular waves. *Ocean Engng* **137**, 394–403.